

# Effect of Cascading the Thermoelectric Peltier Cooler on COP and Cooling Load

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**Abstract:** This paper comprises the Numerical study of various modules of first and second stage of peltier cooler to analyze that how the performance enhances when we shift from single stage to cascade thermoelectric peltier cooler. Multistage thermoelectric coolers have been used from long back as they offer a larger temperature difference than single-stage thermoelectric coolers. Apart from this multistage also helps in improving the COP and cooling load. The COP and heat load were analyzed for various temperature difference and it has been observed that they decrease with increase in temperature difference. For first stage COP ranges from 0.1 to 0.34 and cooling load maximum as 41.9 whereas for second stage COP maximizes to 0.56 and cooling load to 136.45.

## Nomenclature

$Q$  heat flow per unit time, heat power (W)  
 COP coefficient of performance  
 $I$  electric current (A)  
 $Q_h$  Peltier pellet hot side heat flow (W)  
 $Q_c$  Peltier pellet cold side heat flow (W)  
 $P_{in}$  electric power supplied (W)  
 $R_M$  thermal resistance (ohm)  
 $V_{in}$  voltage to the thermoelectric module (volt)  
 $S_M$  Seebeck coefficient (V/K)  
 $DT$  The temperature difference of cold and hot side  
 $T_h$  hot side temperature (K)  
 $T_{12}$  Intermediate temperature between first and second stage in two stage cooler  
 $T_c$  cold side temperature (K)  
 $T_a$  ambient temperature (K)  
 $K_M$  total thermal conductance (W/K)  
 $T$  module temperature for hot and cold side respectively (K)  
 $COP_c$  coefficient of performance of thermoelectric cooling system  
 $S_{mth}$  module Seebeck coefficient at the hot side temperature  $T_h$   
 $S_{mtc}$  module Seebeck coefficient at the cold side temperature  $T_c$   
 $S_1$  Seebeck coefficient of the 1st stage in volts/K  
 $S_2$  Seebeck coefficient of the 2nd stage in volts/K  
 $R_{mth}$  module resistance at the hot side temperature  $T_h$   
 $R_{mtc}$  module resistance at the cold side temperature  $T_c$   
 $R_1$  resistance of the 1st stage in ohms  
 $R_2$  resistance of the 2nd stage in ohms  
 $K_{mth}$  thermal conductance at the hot side temperature  $T_h$   
 $K_{mtc}$  thermal conductance at the cold side temperature  $T_c$   
 $K_1$  thermal conductance of the 1st stage in watts/K  
 $K_2$  thermal conductance of the 2nd stage in watts/K  
 $S_{new}$  Seebeck coefficient for the new module  
 $R_{new}$  electrical resistance of the new module  
 $K_{new}$  thermal conductance of the new module  
 $N_{new}$  number of couples in the new module

$I_{new}$  optimum or maximum current of the new module

**Keywords:** Thermoelectric peltier cooler, Multistaging

## I. INTRODUCTION

Thermoelectric peltier cooler uses the concept of peltier effect which states that when voltage is applied across two junctions of dissimilar electrical conductors, heat is absorbed from one junction and heat is rejected at another junction. The most efficient configuration is that where P and N junction are put electrically in series but thermally in parallel. One side is connected to the heat source and other side to sink. The side facing source is cold side and side facing sink is hot.

Although COP of the thermoelectric peltier coolers is lesser than conventional vapour refrigeration and refrigeration system but continuous research and development work has been done in VCRS system which has led to appreciable improvement in their performance and efficiency. In same manner there is vast scope of improvement in performance of thermoelectric peltier cooler. They are better than VCRS in many aspects. They are compact in size so can be employed where we have space constraint. They do not use harmful refrigerants like Freon for cooling. They do not have any moving part.

## II. LITERATURE REVIEW

Matthieu Cosnier et al (2008) [1] presented a paper on experimental and numerical study of air cooling and heating system based on thermoelectric effect. The cooling power achieved per module was around 50W, with COP ranging from 1.5 to 2, by supplying electricity of 4 ampere and maintaining the temperature difference of around 5°C between the hot and cold sides.

Suwit Jugsujinda et al (2010) [2] conducted a study on analysis of performance of thermoelectric refrigeration system. The thermoelectric refrigeration system (dimension;  $25 \times 25 \times 35 \text{ cm}^3$ ) was constructed by using a thermoelectric cooler (dimension;  $4 \times 4 \text{ cm}^2$ ) and electrical power applied was 40 W. The temperature decreased from 30 °C to 20 °C slowly in around one hour. The maximum COP of refrigerator was 3.0 whereas for cooling system was around 0.65.

Wei He et al (2013) [3] worked out on Theoretical analysis and experimental analysis of a thermoelectric heating and cooling system getting its energy requirement through solar energy. In summer, this device works as a Peltier cooling system running on solar power through photo voltaic modules. The minimum temperature achieved was 17°C, with COP above 0.45.

Riffat and Guoquan et al (2004) [4] carried out comparative analysis of thermoelectric air conditioning system, and conventional systems which includes vapour compression and vapour absorption air conditioning system. Comparison of all of these air conditioning systems was done and a compact air conditioner was constructed.

Astrain, Vian & Dominguez et al [5] carried out an experimental investigation of the COP of a thermoelectric refrigerating system by the optimizing heat rejection. They presented the system based on the principle of a thermo syphon with phase change. In their experiment, a prototype of a thermo syphon system was developed having thermal resistance around 0.11 K/W, dissipating the heat of a Peltier module having dimension of  $40 \times 40 \times 3.9 \text{ mm}$ . They experimental analysis revealed that by using thermo syphon with phase change, an increase of about 32% can be achieved in coefficient of performance.

Shen, Xiao et al (2012) [6] worked out an investigation study on thermoelectric radiant air-conditioning system and presented review paper. In the analysis they employed 51 thermoelectric modules in a panels for indoor cooling, as well as heating by just reversing direction of current. After experimental investigation of their system, they reached the maximum cooling COP of 1.77 by supplying 1.2 and maintaining the colder side temperature to around 20°C.

Virjoghe, Diana et al (2009) [7] carried out a numerical simulation of thermoelectric System. Increase in awareness, for protection of environment, against degradation, rejuvenated interest towards thermoelectric system for efficient utilization of energy. Numerical simulation of various thermoelectric materials has been presented in this paper. These simulations have been carried out by using finite element analysis through ANSYS software.

Maneewan et al (2010) [8] carried out an experimental investigation of compact thermoelectric air conditioning on the basis of thermal compact study. First stage thermoelectric module TEC1-12708 was used for the process of heating and cooling application. Then the COP was calculated and analyzed with respect to time under different conditions.

Manoj and Walke et al (2011) [9] carried out an experimental analysis of thermoelectric air conditioning system for cars. They have been trying to overcome the limitations of existing vapour compression system by replacing with non-conventional air conditioning system working on principle of Peltier and Seebeck effect.

Yadav and Mehta et al (2013) [10] worked out and presented a combined paper on experimental and theoretical analysis of thermoelectric materials and their application. They focused their study on optimization of thermoelectric refrigerator design. The

device was constructed by combining thermoelectric cooler with two element device inserted in n-channel and p-channel of the device to reduce its solid state thermal conductivity.

Manoj Kumar et al (2013) [11] worked out and presented an experimental analysis on green refrigeration and air-conditioning systems. They firstly analyzed various aspects of the existing air-conditioning system, then they compared that with thermoelectric cooling system. Their results concluded that thermoelectric cooling systems generally have efficiency between 5–15% compared to 40–60% achievable by vapour compression system.

Huang. B et al (1999) [12] carried out an experimental study on methods to design the thermoelectric cooling system. They firstly fabricated the thermoelectric system and analyzed it under various conditions. The model simulation revealed that better design of heat sink helps in improving the performance of thermoelectric cooling system.

Jatin Patel, Matik Patel, Jigar Patel, Himanshu Modi [13] worked out numerical study and performed experimental analysis for improvement in COP of Thermoelectric peltier cooling system. They analyzed the performance of thermoelectric cooling system having single stage and multi stage modules. Their result revealed that system using single stage module has lower COP whereas system with multistage (three stages) have higher COP. But the main problem associate with multistage module is their cost. They are expensive compared to the single stage module. With recent developments in field of thermoelectric and nano science, various other thermoelectric materials have been explored which provide higher temperature difference and have lower price. This can help in cot reduction of multistage in near future.

Yamanashi [14] applied heat balance equations in analyzing the coefficient of performance of thermoelectric cooling system by addition of heat exchangers at both side of the device. The balancing equations follow non-equilibrium thermodynamics. For optimizing the two-stage thermoelectric coolers, two different configurations were presented. These configurations are cuboidal pyramidal shaped modules. These arrangements were studied by using COP as an optimization criterion. In the case of the pyramidal shaped module, number of thermocouples between two stages is in ratio of 2.5-3 with lower stage having more modules. The current flowing in the module also follows the same ratio. In the case of cuboidal type module both first and second stage has same number of thermocouples.

Hermes & Barbosa (2012) [15] theoretically analyzed and did the comparison of performance of compact and portable cooling system, which provide cooling through thermoelectric peltier effect, sterling cycle and vapour compression cycle. In their study, Sterling and vapour compression refrigeration system have an overall thermodynamic efficiencies ranging between (14-15)%, whereas the thermoelectric system have about 5%.

Bansal & Martin (2000) [16] carried out investigative study on performance analysis of conventional and non-conventional refrigeration system including vapour compression system, vapour absorption system and thermoelectric system. Their study revealed that the vapour compression refrigeration had highest COP around (2.59), thermoelectric system had (0.69) and absorption refrigeration system had (0.47).

Astrain et al. (2016) [17] aimed at investigation of the effect of various types of heat exchangers (water-air heat exchanger, natural convection heat exchanger, forced convection heat exchanger, liquid cooled heat exchanger) on the efficiency of thermoelectric refrigeration system. Their study concluded that higher value of COP is achievable through optimal designing of the heat exchangers.

### III. NUMERICAL METHODOLOGY

The focus of this study is to investigate the performance of various first stage thermoelectric modules and their comparison with second stage modules

This study has been done for six first stage thermoelectric modules which include:

TEC1-03103, TEC1-07103, TEC1-12705, TEC1-12706, TEC1-12708, TEC1-12710

The nomenclature of the module as example of [TEC1-03103]

TEC- Thermoelectric cooler, 1- first stage, 031- no of couples, 03- for current

And study is done on three second stage thermoelectric module which includes:

TB-2(127-127)1.15, TB-2(127-127)1.15 Bullfinch, TB-2(199-199)0.8

The nomenclature of module as example of [TB-2(127-127)1.15]

TB- product abbreviation for thermoelectric cooler.

127-127 - number of couples in first and second stage of thermoelectric cooler.

1.15- height of the element in the bottom stage.

Various parameters are associated with thermoelectric material and module which are to be considered in mathematical modal. Elements that must be incorporated include seebeck coefficient [SM], Electrical Resistance [RM], and Thermal Conductance [KM]. The specified coefficients are applicable over range of -100<sup>0</sup>C to+150<sup>0</sup>C, and derived from 71 couple 6 ampere module.

1) Seebeck coefficient [SM]- Referred as a function of temperature

$$S_{mth} \text{ or } S_{mtc} = (s_1T) + (0.5 \times s_2 \times T^2) + (0.33 \times s_3 \times T^3) + (0.25 \times s_4 \times T^4) \quad (1)$$

$$SM = (S_{mth} - S_{mtc}) / DT \quad (2)$$

Coefficients for a 71-cpl, 6-amp module

$$s_1 = 1.33450 \times 10^{-2}, s_2 = -5.37574 \times 10^{-5}, s_3 = 7.42731 \times 10^{-7}, s_4 = -1.27141 \times 10^{-9} \text{ for } DT > 0 \quad (3)$$

2) Module resistance [RM]

$$R_{mth} \text{ or } R_{mtc} = (r_1T) + (0.5r_2 \times T^2) + (0.33r_3 \times T^3) + (0.25r_4 \times T^4) \quad (4)$$

$$RM = (R_{mth} - R_{mtc}) / DT \quad (5)$$

Coefficients for 71 couple 6 ampere module

$$r_1 = 2.08317, r_2 = -1.98763 \times 10^{-2}, r_3 = 8.53832 \times 10^{-5}, r_4 = -9.03143 \times 10^{-8} \quad (6)$$

3) Module Thermal conductance [KM]

$$K_{Mth} \text{ or } K_{Mtc} = (k_1T) + (0.5 \times k_2 \times T^2) + (0.33 \times k_3 \times T^3) + (0.25 \times k_4 \times T^4) \quad (7)$$

$$KM = (K_{mth} - K_{mtc}) / DT \quad (8)$$

4) Parameter of conversion for other module

$$S_{new} = (SM \times N_{new}) / 71 \quad (9)$$

$$R_{new} = (RM \times 6 \times N_{new}) / (I_{new} \times 71) \quad (10)$$

$$K_{new} = (KM \times I_{new} \times N_{new}) / (6 \times 71) \quad (11)$$

5) Single Stage module equations

a) Temperature difference(DT) across module in K.

$$DT = T_h - T_c \quad (12)$$

b) (Qc) cooling load by the module in watts

$$Q_c = (S_{new} \times T_c \times I) - (0.5 \times I^2 \times R_{new}) - (K_{new} \times DT) \quad (13)$$

c) The input voltage(Vin) to the module in volts

$$V_{in} = (S_{new} \times DT) + (I \times R_{new}) \quad (14)$$

d) The electrical input power(Pin) to the module in watts

$$P_{in} = V_{in} \times I \quad (15)$$

e) the coefficient of performance(COP)

$$COP = Q_c / P_{in} \quad (16)$$

Here numerical analysis of those modules have been done which have maximum cooling load and maximum COP.

All the modules are having similar maximum COP around 1.61

Here analysis has been done for TEC1-12706

- i)  $DT = 303K - 293K = 10K$
- ii)  $SM = 0.0296V/K, S_{new} = 0.053 V/K$
- iii)  $RM = 1.3525ohm, R_{new} = 1.0708ohm$
- iv)  $KM = 0.2919W/K, K_{new} = 0.5309W/K$
- v)  $I = 6.1 \text{ ampere (from specification)}$
- vi)  $Q_c = (0.053 \times 6.1 \times 293) - (0.5 \times 6.1 \times 6.1 \times 1.0708) - (0.5309 \times 10) = 109.35 \text{ watt}$
- vii)  $V_{in} = (0.053 \times 10) + (6.1 \times 1.0708) = 7.0621 \text{ volt}$
- viii)  $P_{in} = 7.0621 \times 6.1 = 43.0788 \text{ watt}$
- ix)  $COP = 1.6134$

WE are getting maximum cooling load for TEC1-12710

- i)  $DT = 303K - 293K = 10K$
- ii)  $SM = .0296V/K, S_{new} = 0.053V/K$
- iii)  $RM = 1.35251ohm, R_{new} = 0.6467ohm$
- iv)  $KM = 0.2919W/K, K_{new} = 0.8791W/K$
- v)  $I = 8.2$  ampere (from specification)
- vi)  $Q_c = (0.053 \times 8.2 \times 293) - (0.5 \times 8.2 \times 8.2 \times 0.6467) - (0.8791 \times 10) = 115.08$
- vii)  $Q_c(\text{in TOR}) = Q_c / 3500 = 0.0328$

6) Second stage module calculation

There are two different designs of second stage of thermoelectric peltier cooler namely pyramidal shape and cuboidal shape. In the pyramidal shape there is decrease in number of couples in second stage whereas in cuboidal shape both first and second stage have same number of couples. We done the study for cuboidal shape of second stage thermoelectric module

Modeling of cascaded or multi-stage thermoelectric coolers is somewhat more complicated than for single-stage devices. With multi-stage coolers, the temperature between each stage is critically important and module performance cannot be established until each inter stage temperature value is known.

a)  $T_{12} = (T_h + T_c) / 2$  (17)

b) heat pumped by module ( $Q_c$ ) in watts:

$Q_c = (S_2 \times T_c \times I) - (0.5 \times I \times I \times R_2) - (K_2 \times (T_{12} - T_c))$  (18)

c) The input voltage ( $V_{in}$ ) to the module in volt:

$V_{in} = (S_2 \times (T_{12} - T_c)) + (I \times R_2) + (S_1 \times (T_h - T_{12})) + (I \times R_1)$  (19)

d) The input power ( $P_{in}$ ) to the module in watts:

$P_{in} = V_{in} \times I$  (20)

e) coefficient of performance (COP):

$COP = Q_c / P_{in}$  (21)

We got the maximum COP for the module **TB-2(127-127)1.15**

- i)  $DT = 303K - 293K = 10K$
- ii)  $S_1 = 0.053006V/K, S_2 = 0.053006V/K$  ( as both stage have same number of couples)
- iii)  $R_1 = 0.52ohm, R_2 = 0.52ohm$
- iv)  $K_1 = 0.2019W/K, K_2 = 0.2019W/K$
- v)  $I = 8.8$  ampere
- vi)  $T_{12} = (293 + 303) / 2 = 298K$
- vii)  $Q_c = (0.053006 \times 8.8 \times 293) - (0.5 \times 8.8 \times 8.8 \times 0.52) - (0.2019 \times (298 - 293)) = 80.3218$  watt
- viii)  $V_{in} = (0.053006 \times (298 - 293)) + (8.8 \times 0.52) + (0.053006 \times (303 - 298)) + (8.8 \times 0.52) = 6.562$  volts
- ix)  $P_{in} = 6.562 \times 8.8 = 38.0599$
- x)  $COP = 2.11$

Maximum cooling load is achieved for TB-2(199-199)0.8

- i)  $DT = 303K - 293K = 10K$
- ii)  $S_1 = 0.08305V/K, S_2 = 0.08305V/K$  ( as both stage have same number of couples)
- iii)  $R_1 = 0.64ohm, R_2 = 0.64ohm$
- iv)  $K_1 = 0.2782W/K, K_2 = 0.2782W/K$
- v)  $I = 10.2$  ampere
- vi)  $T_{12} = (293 + 303) / 2 = 298$
- vii)  $Q_c = (0.08305 \times 10.2 \times 293) - (0.5 \times 10.2 \times 10.2 \times 0.64) - (0.2782 \times (298 - 293)) = 213.538$  watt
- viii)  $Q_c(\text{in TOR}) = 213.538 / 3500 = 0.061$

From this analysis it can be inferred that higher COP and cooling load can be achieved by multi staging the thermoelectric module and we can even achieve higher COP with third and fourth stage.

#### IV. RESULTS & DISCUSSIONS

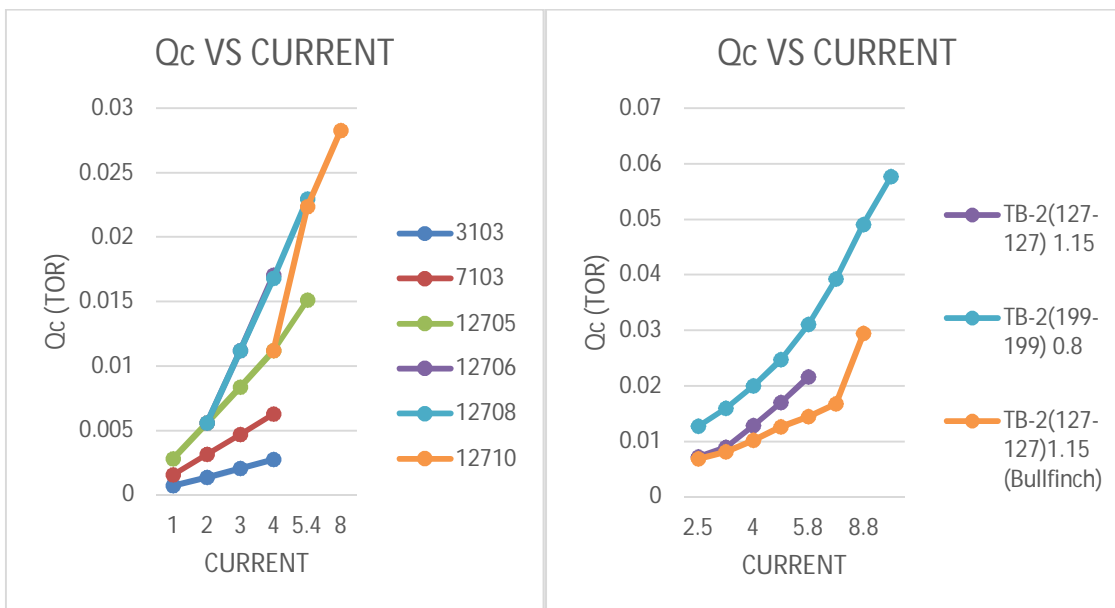


Figure.1- Qc(TOR) Vs Current

Figure.2- Qc(TOR) Vs Current

- 1) From figure 1 and 2, it can be interpreted that the cooling load increase with increase in the current.
- 2) Cooling load of the module increase due to thermal conductivity of the module.
- 3) For first stage we are getting maximum Qc(TOR) for all modules and, second stage maximum Qc(TOR) for TB-2(199-199)0.8.

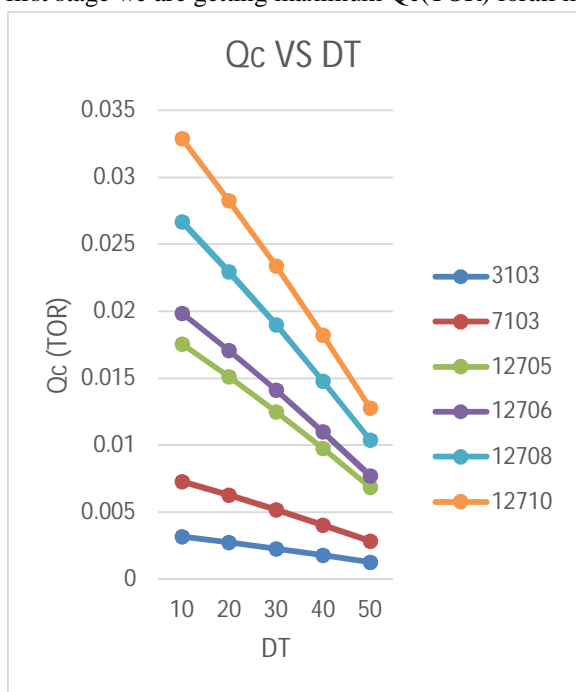


Figure.3- Qc Vs dt

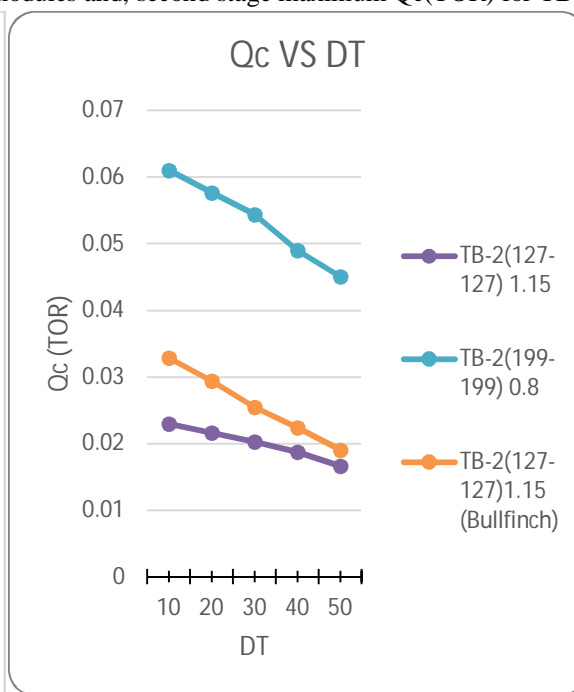


Figure.4- Qc Vs dt

- 4) In all the modules cooling load decrease with increase in the temperature difference between hot and cold side.
- 5) As the temperature difference increases the efficiency of heat exchanger system decreases. The efficiency of heat exchanger strongly influence performance parameters including COP. That's why cooling load as well COP decrease with increase with temperature difference.



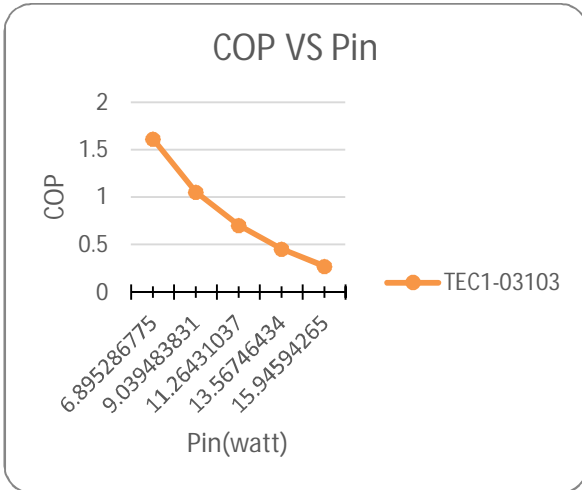


Figure.5- COP Vs Pin

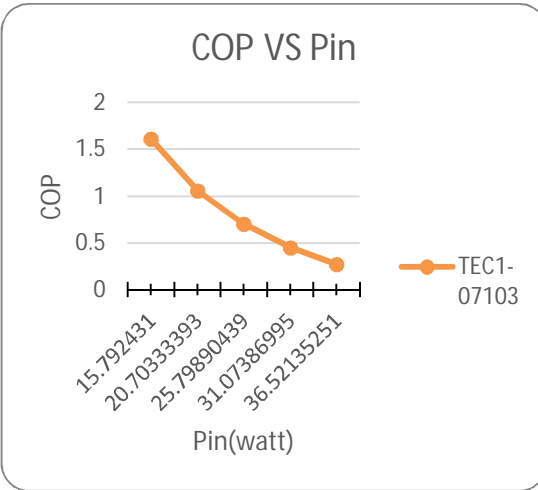


Figure.6- COP Vs Pin

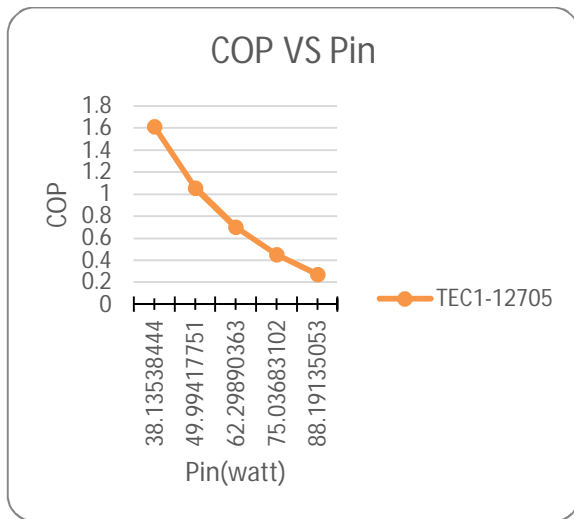


Figure.7- COP Vs Pin

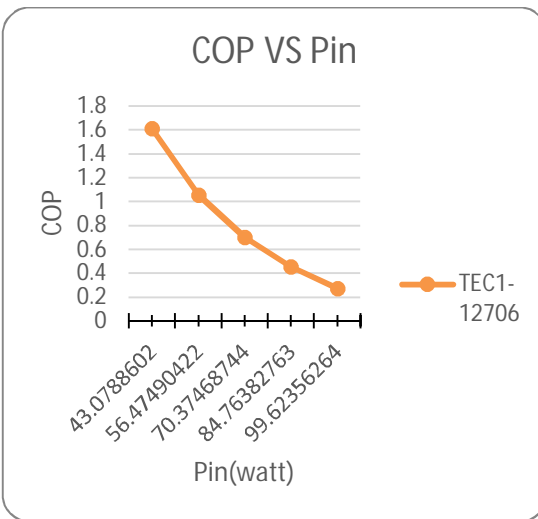


Figure.8- COP Vs Pin

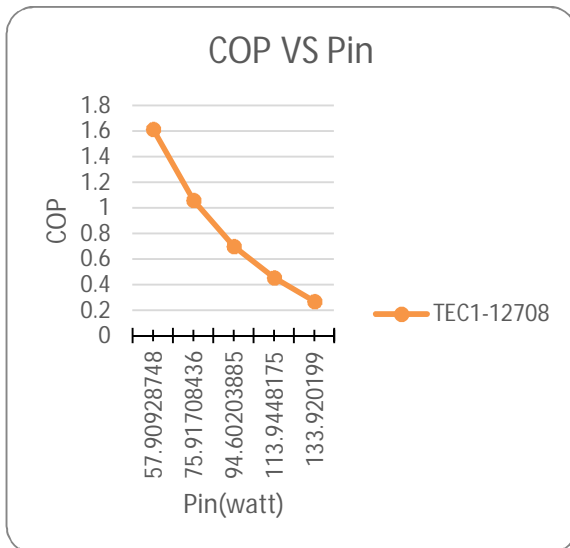


Figure.9- COP Vs Pin

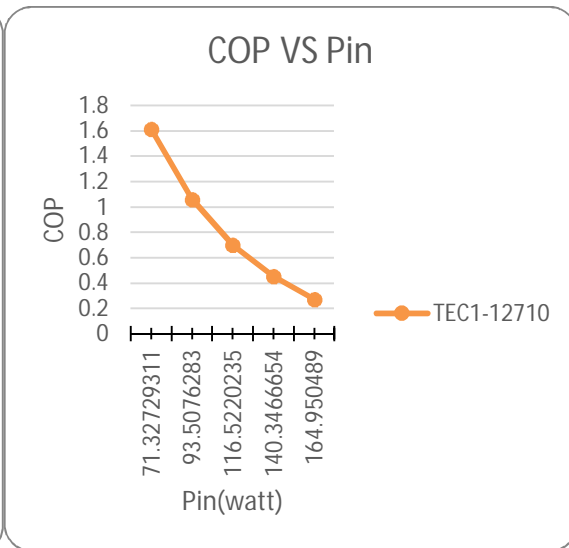


Figure.10- COP Vs Pin

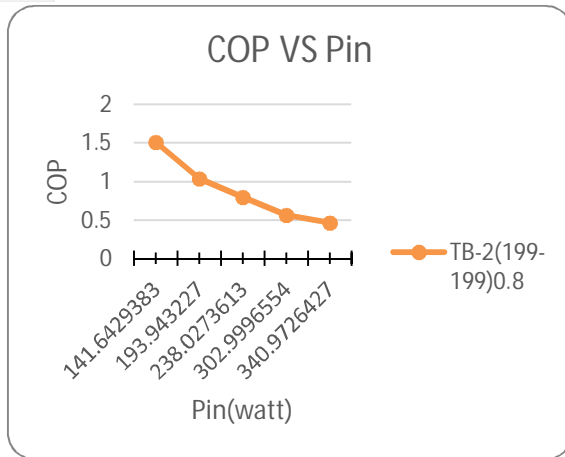


Figure.11- COP Vs Pin

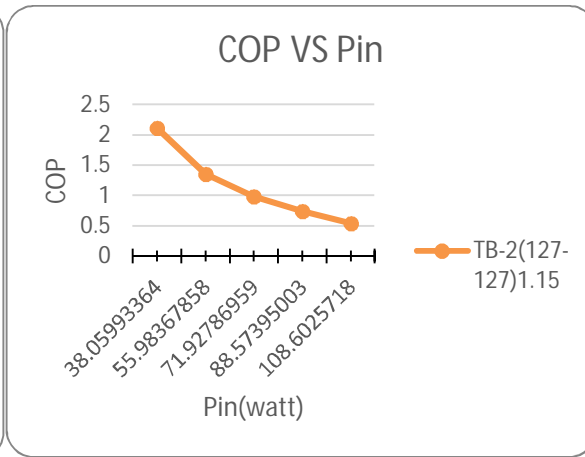


Figure.12- COP Vs Pin

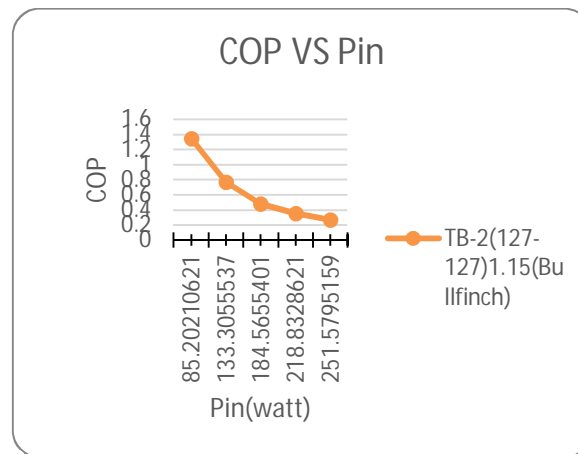


Figure.13- COP Vs Pin

- 6) From figure 5 to 13 we observe the variation of COP with input power.
- 7) COP increase with decrease in input power so we can say less power is required to achieve higher power.
- 8) In first stage we get maximum COP for all module is similar and around 1.61
- 9) In second stage we are getting maximum COP for TB-2(127-127)1.15, interpreted from figure 12, which is 2.1.

## V. CONCLUSION

From the above analysis it can be concluded that cascading or multi staging helps in enhancing various parameters like COP and cooling load of thermoelectric peltier cooler.

- A. Maximum value of COP is 1.61 for a first stage whereas it is around 2.11 for the second stage. Around 31% improvement in COP can be obtained by multi staging.
- B. As COP increases with lesser power input means rate of consumption of energy also decreases with COP increase.
- C. The maximum value of cooling load for first stage is 115.082 whereas for second stage it is around 213.538.
- D. The graph indicates that both the COP and cooling load decrease as the temperature difference increases.
- E. Where larger temperature difference is required COP can be greatly improved by using multistage or cascade thermoelectric peltier module.
- F. One problem with the multi staging is that price of the module are high. But researches are going on in the field of thermoelectric and nanoscience to make material with good thermal performance at a cheaper price.
- G. There is vast scope of further development in design of the thermoelectric cooler setup. As an alternative for normal axial fan used, if a blower fans is selected, the cooling system would provide better airflow. Advancement in design for hot and cold side heat sink can effectively help in achieving better performance parameters.



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