Design and Structural Analysis of Turbine Blades

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Abstract: A steam turbine is a mechanical device that concentrates warm vitality from pressurized steam and changes over it into rotating movement. An arrangement of calculated and molded edges organized on a rotor through which steam is passed to produce rotational vitality. Moving liquid follows up on the cutting edges with the goal that they move and bestow rotational vitality to the rotor. The cutting edges are outlined in such an approach to deliver most extreme rotational vitality by coordinating the stream of the steam along its surface. The sharp edges are made at particular points keeping in mind the end goal to consolidate the net stream of steam over it to support its. The sharp edges might be of stationary or settled and turning or moving sorts, and shaft is intended to work in outrageous conditions, hear it needs to endure the temperature which is originating from the steam and loads (weight and radiating power) of the edges gathering and other get together parts.
The point of the work is to outline a steam turbine sharp edge utilizing 3D displaying programming CATIA V5 R20 and reproducing basic investigation utilizing ANSYS 15.0 on edge by applying diverse materials properties. Directing the investigation of stresses creating on cutting edge, mode state of the edge conduct is found and utilizing examination comes about the best material for sharp edge is proposed.
Keywords: steam turbine, structural analysis, catia V5 R20, mode shapes.

I. INTRODUCTION

A turbine is a rotating mechanical gadget that concentrates vitality from a liquid stream and changes over it into valuable work. A turbine is a turbo machine with no less than one moving part called a rotor get together, with edges joined. Moving liquid follows up on the edges with the goal that they move and confer rotational vitality to the rotor. Turbines are real prime movers in warm power stations. The primary parts of straightforward motivation steam turbine are rotor, sharp edges and spouts. Turbine cutting edge is presented to different loads, for example, warm, inactivity, and bowing and may flop because of various elements like Stress-Corrosion Cracking, High-Cycle Fatigue, Corrosion-Fatigue Cracking, Temperature Creep Rupture, Low-Cycle Fatigue, consumption, and so on. The issue with cutting edges is the Excessive Stresses Resonance because of vibration Operating ecological impacts. Always expanding requests of superior together with unwavering quality of operation, long life and lightweight. Unreasonable burdens are the aggregate worry at any area of the cutting edge is, whole of the diffusive strain, divergent twisting, relentless steam bowing and the substituting bowing. The abundancy of rotating twisting relies upon the dynamic bowing power, damping factor and the full recurrence.

Exhaustion is caused by visit begin/stop operations, warm cycling and successive water slugging or water washing, because of deficient water waste in the packaging and can make disappointments inside couple of hundreds a couple of thousands anxiety cycle. Vibration is essential in planning turbine sharp edges/plate since full vibratory anxieties, maintained over some stretch of time, can cause weakness disappointments. Cutting edges represents a most exceedingly awful vibratory weariness issue since it is specifically presented to an extensive variety of streamlined excitation and disappointment can come about when any of the accompanying coordinating happens.

II. LITERATURE REVIEW

[1] John. V, T. Ramakrishna was explored one outline and investigation of Gas turbine cutting edge, CATIA is utilized for plan of strong model and ANSYS programming for examination for F.E. display produced, by applying limit condition, this paper additionally incorporates particular post preparing and life appraisal of sharp edge. How the program makes successful utilization of the ANSYS pre-processor to work complex geometries of turbine cutting edge and apply limit conditions. [2] Subramanyam Pavuluri, Dr. A. Siva Kumar was researched on outline of high weight steam turbine sharp edge tends to the issue of steam turbine proficiency. A particular concentrate on airfoil profile for high-weight turbine cutting edge, and it assesses the adequacy of certain Chromium and Nickel in opposing jerk and crack in turbine sharp edges. The proficiency of the steam turbine is a key factor in both

the natural and monetary effect of any coal-let go control station. In light of the exploration introduced adjustments to high-weight steam turbine cutting edges can have made to build turbine proficiency of the turbine. The outcomes and decisions are introduced for a concerning the solidness issues experienced with steam turbine cutting edges. The most extreme operational Von Mises Stresses are inside the yield quality of the material yet the misshapening is relatively better for material Inconel 600 (Chromium Nickel). Adjusted answers for Steam turbine cutting edge esteems to machines to amplify the decrease in life cycle costs, effectiveness, andimprove dependability. [3] Sanjay Kumar was researched on crawl life of turbine cutting edge. Latency stack is the consistent load that will cause crawl disappointment. Crawl is a rate subordinate material nonlinearity in which material keeps on disfiguring in nonlinear design even under consistent load. This wonder is dominating in parts, which presented to high temperatures. By concentrate the crawl wonder and foreseeing the crawl life of the part, we can evaluate its plan life. The fundamental target is to foresee the crawl life of the straightforward motivation steam turbine cutting edge, and to give the FEM approach for crawl investigation. The examination of turbine sharp edge for various burdens, which demonstrates that the most extreme anxieties, prompted for each situation. These anxieties are inside yield breaking point of the material and won't experience plastic twisting amid operation result is discovered that, crawl life diminishes as the anxiety esteem increments. Consequently, by diminishing the anxiety esteem in the segment we can build its crawl life. This was being accomplished by adjusting the sharp edge plan. [4]Jackson, et.al made a preparatory plan of a 50 meters in length cutting edge, two variants one of fiber glass and one with carbon composite was utilized to test the cost and thickness of cross segments was changed so as to enhance auxiliary productivity. The streamlined execution was made utilizing computational strategies and the calculations were anticipated utilizing spotless and ruined surface Karam and Hani upgraded utilizing the factors as cross segment territory, span of gyration and the harmony length, the ideal outline is for most extreme common recurrence. The improvement is finished utilizing multi-dimensional hunt strategies. The outcomes had demonstrated the method was productive. [5] K.J. Johansen, and Sorensen N.N, altered the tip of the rotors to winglet to enhance the streamlined execution of turbine rotors and to make them less touchy to wind blasts. Maughmer, M.D, tilted the edge tip to the impact of winglets which diminishes the prompted drag of the edge by changing the downwash appropriation, subsequently expanding the power creation.

III. MODELING

A. Introduction to CAD

Computer-aided design (CAD), also known as computer-aided design and drafting (CADD), is the use of computer technology for the process of design and design-documentation. Computer Aided Drafting describes the process of drafting with a computer. CADD software, or environments, provide the user with input-tools for the purpose of streamlining design processes; drafting, documentation, and manufacturing processes. CADD output is often in the form of electronic files for print or machining operations.
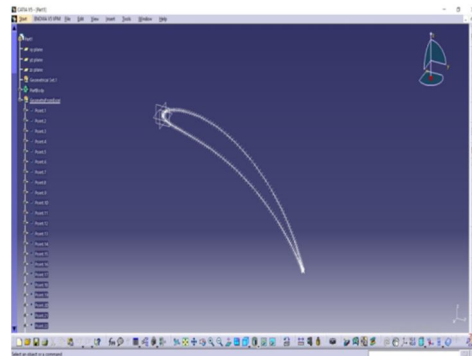

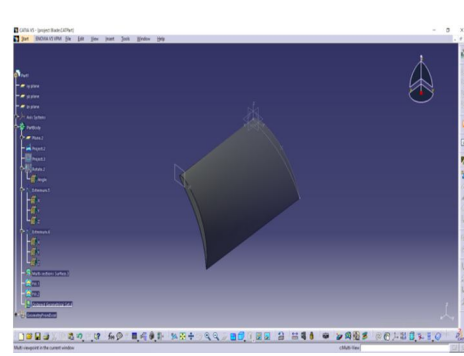Fig. 3.1 CATIA V5 R20 Importing Blade Profile
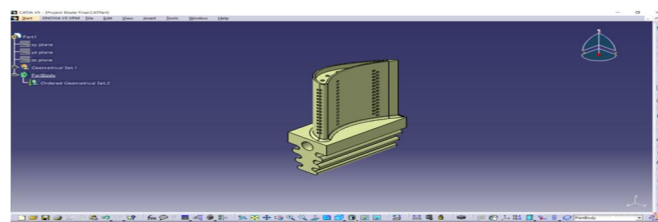

Fig. 3.2 Forming the Curved Blade Surface

Fig. 3.3 Forming the Blade root and Cooling Holes

B. Introduction To FEA

Finite Element Analysis (FEA) is a computer-based numerical technique for calculating the strength and behavior of engineering structures. It can be used to calculate deflection, stress, vibration, buckling behavior and many other phenomena. It can be used to analyze either small or large-scale deflection under loading or applied displacement. It can analyze elastic deformation, or "permanently bent out of shape" plastic deformation. The computer is required because of the astronomical number of calculations needed to analyze a large structure. The power and low cost of modern computers has made Finite Element Analysis available to many disciplines and companies

1) Steps Involved Finite Element Analysis:

a) Divide the continuum into a finite number of sub regions (or elements) of simple geometry such as line segments, triangles, quadrilaterals (Square and rectangular elements are subset of quadrilateral), tetrahedrons and hexahedrons (cubes) etc.
b) Select key point on the elements to serve as nodes where conditions of equilibrium and compatibility are to be enforced
c) Assume displacement functions within each element so that the displacements at each generic point are depending upon nodal values.
d) Satisfy strain displacement and stress – strain relationship within a typical element
e) Determine stiffness and equivalent nodal loads for a typical element using work or energy principles.
f) Develop equilibrium equations for the nodes of the discretized
g) Continuum in terms of the element contributions
h) Solve the equilibrium for the nodal displacements.

Table 1 Material properties of inconel 600 & structural steel

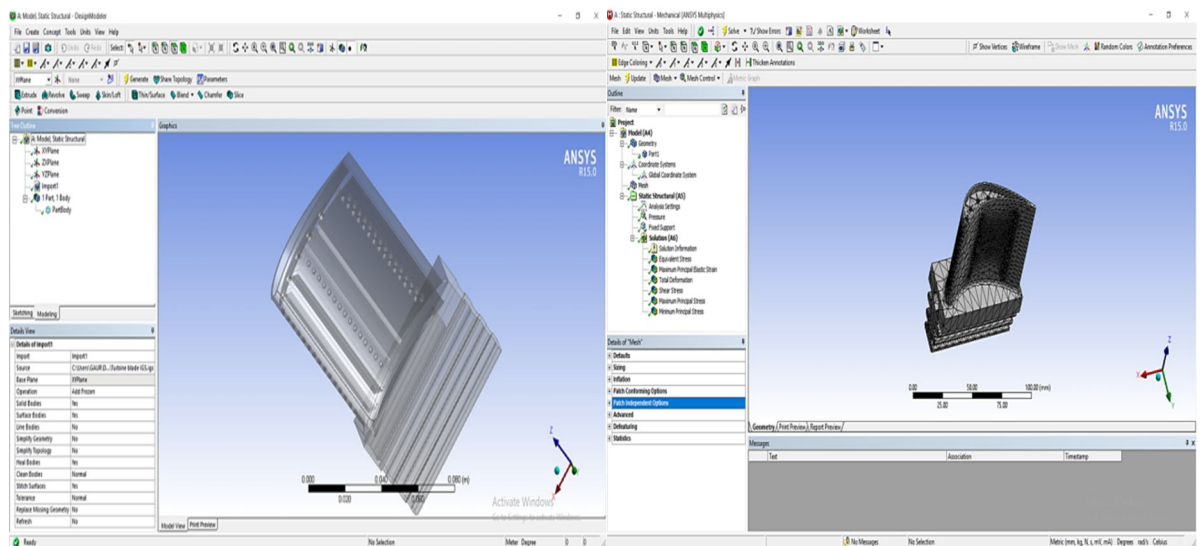| Mechanical Properties	Inconel 600	Structural Steel
Density (Kg/m^3)	8800	7850
Modulus of Elasticity (GPa)	214	201
Yield Strength (Pa)	3.1×10^8	2.3×10^8
Ultimate Strength (Pa)	5.8×10^8	4.6×10^8
Poisson Ratio	0.324	0.3
Hardness Brinell	172	149
Shear modulus (Pa)	$8.08 \times e^{10}$	3.63×10^{10}
Compressive Strength (Pa)	$7.69 \times e^{10}$	7.69×10^{10}

Fig. 3.4 Importing blade geometry to Ansys software.   Fig. 3.5 Ansys Meshing
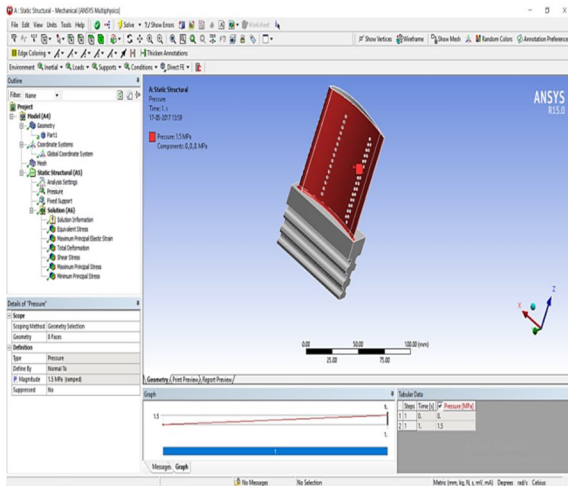
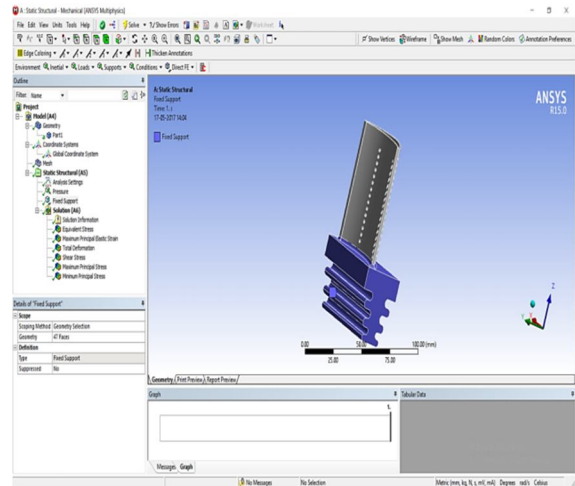Fig. 3.6 Applying Forces   Fig. 3.7 Fixed Support

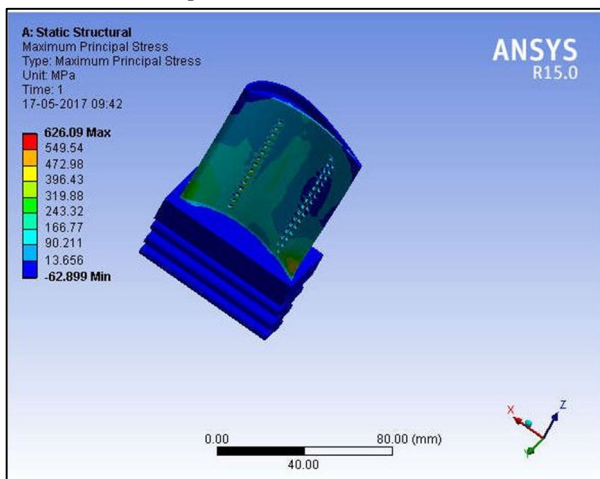## IV. ANSYS RESULTS

A. Maximum Principal Stress

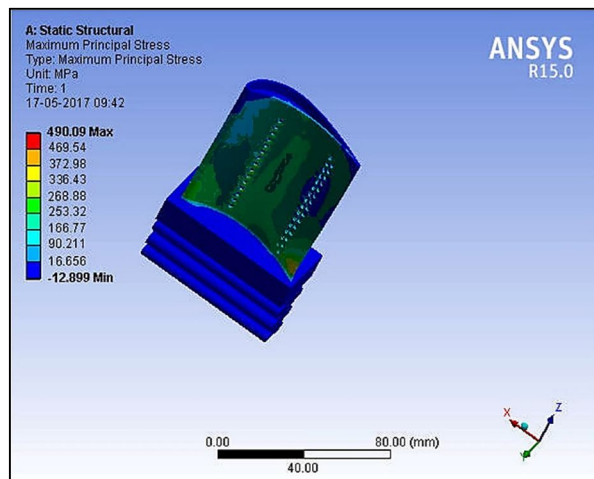Fig. 4.1 Maximum Principal Stress for Structural Steel   Fig. 4.2 Maximum Principal Stress for Inconel 600
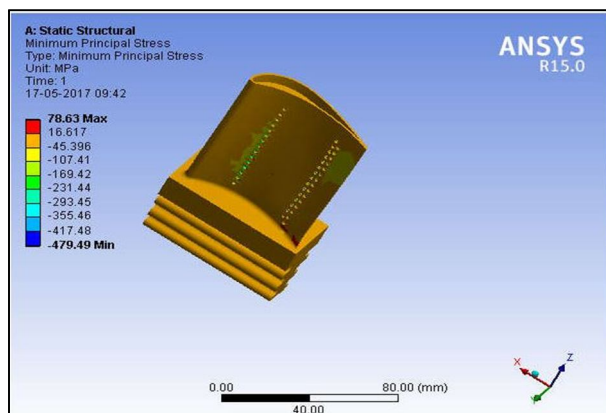
B. Minimum Principal Stress
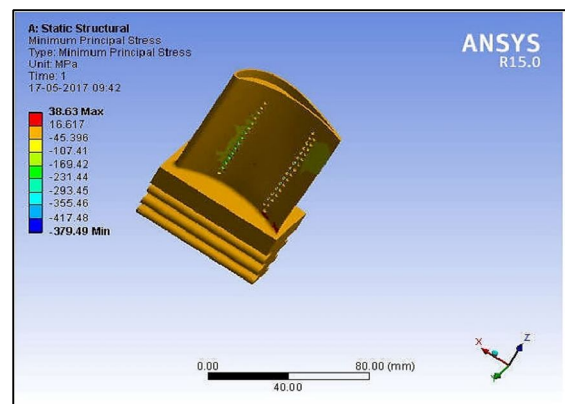

Fig. 4.3 Minimum Principal Stress for Structural Steel   Fig. 4.4 Minimum Principal Stress for Inconel 600

C. *Von Mises Stress*
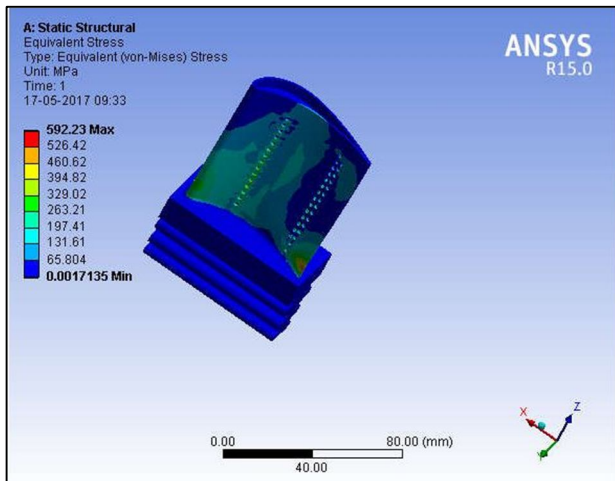

Fig. 4.5 Equivalent Stress for Structural Steel

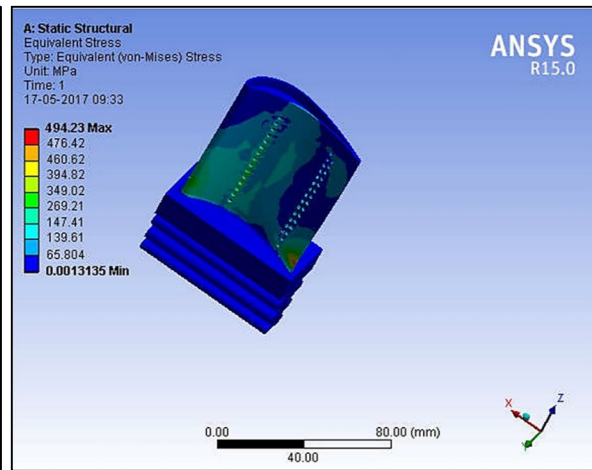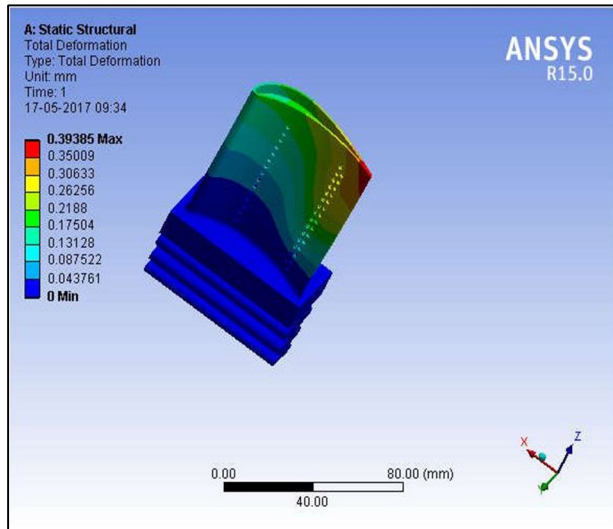Fig. 4.6 Equivalent Stress for Inconel 600

D. *Total Deformation*
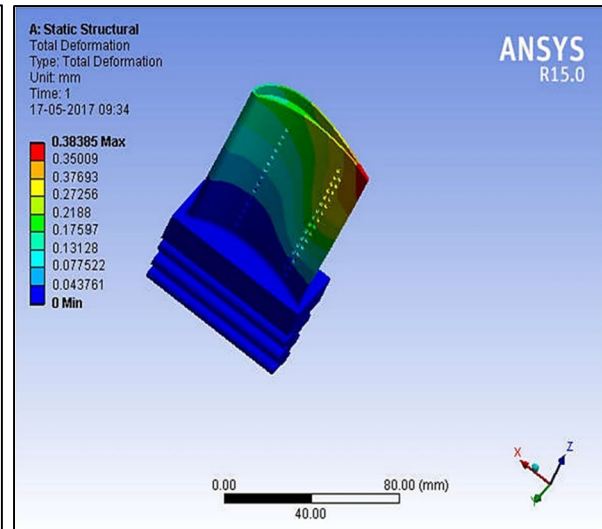

Fig. 4.7 Total Deformation for Structural Steel

Fig. 4.8 Total Deformation for Inconel 600
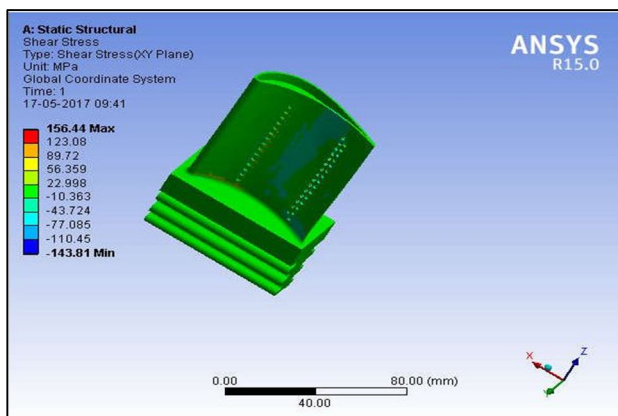
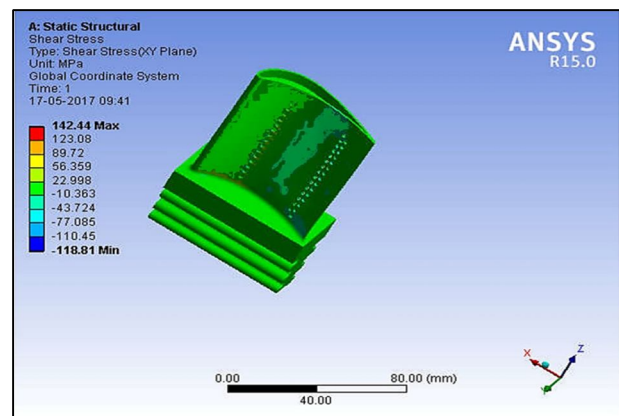E. *Shear Stress*

Fig. 4.9 Shear Stress for Structural Steel

Fig. 4.10 Shear Stress for Inconel 600

F. *Maximum Principal Elastic Strain*
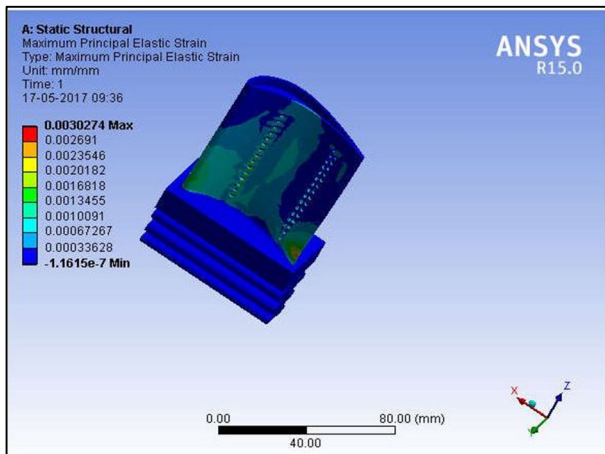

Fig. 4.11 Max. Principal Elastic Strain for Structural Steel
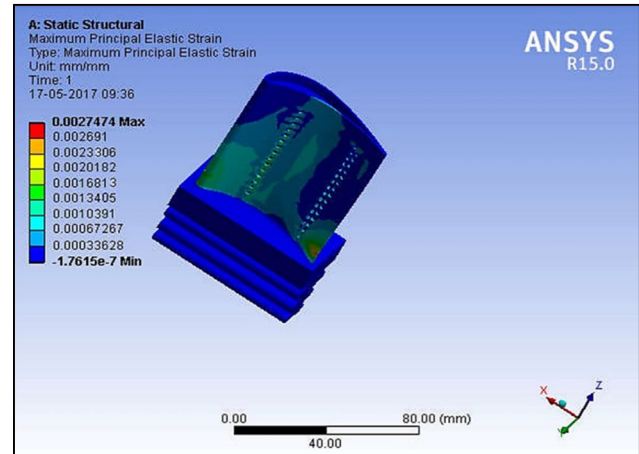
Fig. 4.12 Minimum Principal Strain for Inconel 600

V.   CONCLUSION

We have analyzed previous designs and generals of turbine blade to do further optimization, Finite element results for free standing blades give a complete picture of structural characteristics, which can have utilized for the improvement in the design and optimization of the operating conditions. From the analysis, it can be seen that the stresses developing on blade, mode shape of the blade behavior is found for both Structural Steel and Inconel 600. The results obtained suggest that the best material is Inconel 600. The conclusions are presented for a concerning the durability problems experienced with steam turbine blades. The maximum operational Von Mises Stresses are within the yield strength of the material but the deformation is comparatively lower for Inconel 600 (Chromium Nickel).

Table 2 ANSYS Result Comparison between Structural steel and Inconel 600

MECHANICAL PROPERTIES	MATERIALS	
	Structual steel	Inconel 600
Maximum Principal Stress (MPa)	626.09	490.09
Minimum Principal Stress(MPa)	78.63	38.63
Von Mises Stress (MPa)	592.23	494.23
Total Deformation (mm)	0.39385	0.38385
Shear Stress (MPa)	156.44	142.44
Maximum Principal Elastic Strain	0.0030274	0.0027474

REFERENCES

[1]   John. V, T. Ramakrishna, ″The Design and Analysis of Gas Turbine Blade″, International Journal of Advanced Engineering Research and Studies, VOL 2, ISSUE I, OCT-DEC, 2012 page no. 53-55

[2]   Subramaniam PAVULURI, DR. A Siva Kumar, ″ Experimental Investigation on Design of High Pressure Steam Turbine Blade″, International Journal of Innovative Research in Science, Engineering and Technology 2, ISSUE 5, MAY 2013, ISSN:2319-8753

[3]   Dunham, J. A parametric method of Turbine Blade Profile design, ASME Paper 74-GT-119, 1974

[4]   S. M. Sanjay Kumar, ″Creep Life Prediction of SteamTurbine Blade Using Finite Element Method″International Journal of Advanced EngineeringResearch and Studies, VOL. I, NO.2, PP. 95-98

[5]   Avinash V. Sarlashkar, Mark L. Redding,"AnANSYS-Based Turbine Blade Analysis System ImpactTechnologies", LLC, ROCHESTER, NY 14623, U.S.A

[6] A Review on Analysis of Low Pressure Stage of Steam Turbine Blade with FEA (ANSYS Software), IJSRD/Vol. 1/Issue 10/2013/0003

[7] David Gordon Wilson; 1991 "The Design ofHigh-Efficiency Turbo machinery and Gas Turbines", pages 487-492, published by the MIT Press, Cambridge, Massachusetts, 1984, 5th printing 1991

[8] Johansen, J. and Sorensen, N.N., "Numerical Analysis of Winglets on Wind Turbine Blades using CFD", EWEC 2007 Conference proceedings, Madrid, Spain, 2007.

[9] Karam Y, Hani M, "Optimal frequency design of wind turbine blades", Journal of Wind Engineering and Industrial Aerodynamics 90 (2002) 961–986

[10] K.J. Jackson, et al., "Innovative design approaches for large wind turbine blades", 43rd AIAA Aerospace Sciences Meeting and Exhibit 10 - 13 January 2005, Reno, Nevada

[11] Maughmer, M.D, "The Design of Winglets for High-Performance Sailplanes", AIAA 2001-2406, AIAA Applied Aerodynamics Conference, 19th, Anaheim, CA, June 11-14, 2001

[12] M. Jureczko, M. Pawlak, A. Mezyk, "Optimization of wind turbine blades ", Journal of Materials Processing Technology 167 (2005) 463–471.

[13] Philippe Giguere and Selig, "Blade Geometry Optimization For The Design Of Wind Turbine Rotors" AIAA-2000-0045.

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